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Corrigendum

Corrigendum to 'Construction of new enriched beam models accounting for cross-section deformation and pinching'*



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The beam model introduced for thick beams undergoing large motions account for non-classical effects such as warping, out- and in-plane deformations. Although the introduction of a set of directors attached to the beam section described in equations (1) to (5) of the paper is a well-adopted notion in the literature since the early work of Antman [5], the beam kinematics has been properly formulated in an achieved form in the more recent contribution [D. Durville, Computational Mechanics, 49, Issue 6, pp 687–707, 2012], using the same notations. We acknowledge that this work is an important contribution in the field of nonlinear beam mechanics.

The expression of the Green-Lagrange strain includes few misprints and shall be corrected as follows, starting from the transformation gradient:

$$\begin{split} \mathbf{F} &= \mathbf{d}_{\alpha} \otimes \mathbf{e}_{\alpha} + \left(\boldsymbol{\xi}_{\alpha} \mathbf{d'}_{\alpha} + \mathbf{r'} \right) \otimes \mathbf{e}_{3} \\ \mathbf{F}^{T} \mathbf{F} &= \left(\mathbf{e}_{\alpha} \otimes \mathbf{d}_{\alpha} \right) \left(\mathbf{d}_{\beta} \otimes \mathbf{e}_{\beta} \right) + \left(\mathbf{e}_{\alpha} \otimes \mathbf{d}_{\alpha} \right) \left[\left(\boldsymbol{\xi}_{\beta} \mathbf{d'}_{\beta} + \mathbf{r'} \right) \otimes \mathbf{e}_{3} \right] \\ &+ \left[\mathbf{e}_{3} \otimes \left(\boldsymbol{\xi}_{\beta} \mathbf{d'}_{\beta} + \mathbf{r'} \right) \right] \left(\mathbf{d}_{\alpha} \otimes \mathbf{e}_{\alpha} \right) + \left[\mathbf{e}_{3} \otimes \left(\boldsymbol{\xi}_{\beta} \mathbf{d'}_{\beta} + \mathbf{r'} \right) \right] \left[\left(\boldsymbol{\xi}_{\beta} \mathbf{d'}_{\beta} + \mathbf{r'} \right) \otimes \mathbf{e}_{3} \right] \\ &= \left(\mathbf{d}_{\alpha} \cdot \mathbf{d}_{\beta} \right) \mathbf{e}_{\alpha} \otimes \mathbf{e}_{\beta} + \left(\boldsymbol{\xi}_{\beta} \mathbf{d'}_{\beta} \cdot \mathbf{d}_{\alpha} + \mathbf{r'} \cdot \mathbf{d}_{\alpha} \right) \left(\mathbf{e}_{\alpha} \otimes \mathbf{e}_{3} + \mathbf{e}_{3} \otimes \mathbf{e}_{\alpha} \right) \\ &+ \left(\mathbf{r'} \cdot \mathbf{r'} + 2 \boldsymbol{\xi}_{\alpha} \mathbf{d'}_{\alpha} \cdot \mathbf{r'} + \boldsymbol{\xi}_{\alpha} \boldsymbol{\xi}_{\beta} \mathbf{d'}_{\alpha} \cdot \mathbf{d'}_{\beta} \right) \mathbf{e}_{3} \otimes \mathbf{e}_{3} \end{split}$$

$$\begin{split} \mathbf{E} &= \frac{1}{2} \big(\mathbf{F}^T \mathbf{F} - \mathbf{I} \big) \\ &= \frac{1}{2} \begin{pmatrix} \mathbf{d}_1 \cdot \mathbf{d}_1 - 1 & \mathbf{d}_1 \cdot \mathbf{d}_2 & \xi_1 \mathbf{d}'_1 \cdot \mathbf{d}_1 + \xi_2 \mathbf{d}'_2 \cdot \mathbf{d}_1 + \mathbf{r}' \cdot \mathbf{d}_1 \\ \mathbf{d}_1 \cdot \mathbf{d}_2 & \mathbf{d}_2 \cdot \mathbf{d}_2 - 1 & \xi_1 \mathbf{d}'_1 \cdot \mathbf{d}_2 + \xi_2 \mathbf{d}'_2 \cdot \mathbf{d}_2 + \mathbf{r}' \cdot \mathbf{d}_2 \\ \xi_1 \mathbf{d}'_1 \cdot \mathbf{d}_1 & \xi_1 \mathbf{d}'_1 \cdot \mathbf{d}_2 & \mathbf{r}' \cdot \mathbf{r}' + 2\xi_1 \mathbf{d}'_1 \cdot \mathbf{r}' + 2\xi_2 \mathbf{d}'_2 \cdot \mathbf{r}' \\ + \xi_2 \mathbf{d}'_2 \cdot \mathbf{d}_1 + \mathbf{r}' \cdot \mathbf{d}_1 & + \xi_2 \mathbf{d}'_2 \cdot \mathbf{d}_2 + \mathbf{r}' \cdot \mathbf{d}_2 & + \xi_1^2 \mathbf{d}'_1 \cdot \mathbf{d}'_1 + 2\xi_1 \xi_2 \mathbf{d}'_1 \cdot \mathbf{d}'_2 + \xi_2^2 \mathbf{d}'_2 \cdot \mathbf{d}'_2 - 1 \big) \end{split}$$

We would also like to underline that the strain quantities $\mathbf{d}_i.\mathbf{d}'_j$ exist for rigid sections; they describe the coupling between transverse and longitudinal deformations and they can be interpreted as representing the beam torsion. Warping involves a nonlinear kinematic model, and it is not possible for linear kinematic models since the beam sections remain planar.

The authors would like to apologise for any inconvenience caused.

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